

Principal Stress & Maximal Shear Stress

2-Dimensional

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\tau_{max} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_{y'} = \frac{\sigma_x + \sigma_y}{2} - \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\left(\frac{\sigma_x - \sigma_y}{2}\right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\theta = \frac{1}{2} \tan^{-1} \left(\frac{2\tau_{xy}}{\sigma_x - \sigma_y} \right)$$

$$I_{xx} = \int y^2 dA, I_{yy} = \int x^2 dA, I_{xy} = \int xy dA, I_{x'x'} = I_{xx} \cos^4 \theta - 2I_{xy} \cos^2 \theta \sin 2\theta + I_{yy} \sin^4 \theta + I_{xy} \sin 4\theta$$

3-Dimensional

Determinants Equations (Stress Invariants)

$$I_1 = \sigma_x + \sigma_y + \sigma_z$$

$$I_2 = \sigma_x \sigma_y + \sigma_x \sigma_z + \sigma_y \sigma_z - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{xz}^2$$

$$I_3 = \sigma_x \sigma_y \sigma_z + 2\tau_{xy} \tau_{yz} \tau_{xz} - \sigma_x \tau_{xy}^2 - \sigma_y \tau_{yz}^2 - \sigma_z \tau_{xz}^2 - 3\tau_{xy} \tau_{yz} \tau_{xz}$$

$$I_1^3 - 3I_1 I_2 + 3I_3 = 0 \Rightarrow \sigma_1, \sigma_2, \sigma_3$$

Principle stress of 1, 2, 3, where

$$\sigma_{max} = \frac{I_1 + \sqrt{I_1^2 - 3I_2}}{3}$$

See ex. for principal direction

$$\sigma_{min} = \frac{I_1 - \sqrt{I_1^2 - 3I_2}}{3}$$

$$\sigma_{int} = \frac{I_1 - \sqrt{I_1^2 - 3I_2}}{3}$$

Centroidal Bending

$$\sigma_{max} = \sum \frac{M y}{I} = \frac{M}{I} c$$

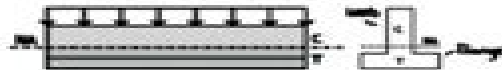
c is distance from NB to point of interest; it differs for circular shaft

I is NB with the MCT neutral axis

bending stress will be MCT (symptical) axis

MCA is neutral (symptical) stress distribution

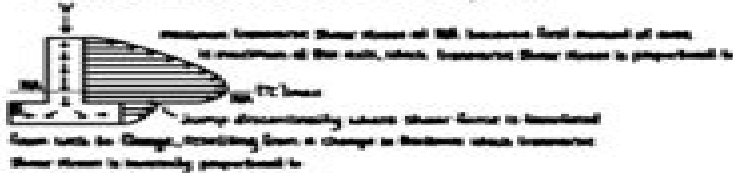
tensile bending → positive, compression bending → negative



Shear Flow

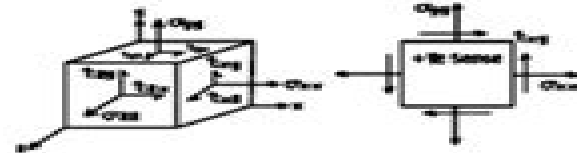
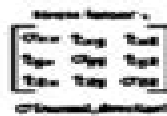
$$v = \frac{VQ}{I}$$

Q → Area, for rectangular along axis but I & Q are with respect to I & Q will be MCT axis opposite to V (another axis (symptical))



Torsion

$$\tau = \frac{T \rho}{J}$$



Case Sectional Properties

$$Centroid: \bar{y} = \frac{\sum y_i A_i}{\sum A_i}$$

Second Moment of Area

$$I = \sum I_{self} + \sum A_i d_i^2 \text{ only use if other axis is not coincident}$$

$$d_i = (y_i - \bar{y})$$

If I_{self} have (d) in their perspective of y_i-axis

If I_{self} " " z_i-axis

Area Moment of Inertia

$$J = \sum (I_{self})^2 \text{ if bar is coincident to shaft; hollow use } d_o^4 - d_i^4$$

$$I_x = \sum (I_{self})^2 \text{ if bar is coincident to beam}$$

First Moment of Area

moment of area above or below point of interest

$$Q = A \bar{y}$$

where A is the remaining case sectional area above or below that point

and \bar{y} is the distance between the centroid of that area to the centroid of the entire case section

2-Dimensional Plane Stress & Strain

Stress

(1) — one dimension compared to other two dimension is very small

(2) — plate or a shell, stress with large constant

(3) — shell has to be in the plane of the plate, etc., same way it have etc.

Strain

(1) — one dimension compared to other two dimension is very large

(2) — symmetric loading condition

Stress-Strain

A: point of interest

$$\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ C_{21} & C_{22} & C_{23} \\ C_{31} & C_{32} & C_{33} \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{21} & E_{22} & E_{23} \\ E_{31} & E_{32} & E_{33} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & \nu_{12} & \nu_{13} \\ \nu_{21} & E_{22} & \nu_{23} \\ \nu_{31} & \nu_{32} & E_{33} \end{bmatrix}$$

Stress-Strain Relationship

$$E_{xx} = \frac{1}{\nu} \left(\frac{E}{1 + \nu} \right) \left(\epsilon_x + \nu \epsilon_y + \nu \epsilon_z \right)$$

$$E_{yy} = \frac{1}{\nu} \left(\frac{E}{1 + \nu} \right) \left(\nu \epsilon_x + \epsilon_y + \nu \epsilon_z \right)$$

$$E_{zz} = \frac{1}{\nu} \left(\frac{E}{1 + \nu} \right) \left(\nu \epsilon_x + \nu \epsilon_y + \epsilon_z \right)$$

$$\tau_{xy} = \frac{E}{2(1 + \nu)} \gamma_{xy}, \tau_{yz} = \frac{E}{2(1 + \nu)} \gamma_{yz}, \tau_{xz} = \frac{E}{2(1 + \nu)} \gamma_{xz}$$

C = G(2G-1)

Plane Stress Condition

$$E_{xx} = \frac{E}{1 + \nu} \left(\epsilon_x + \nu \epsilon_y \right)$$

$$E_{yy} = \frac{E}{1 + \nu} \left(\nu \epsilon_x + \epsilon_y \right)$$

$$E_{zz} = -\nu \frac{E}{1 + \nu} \left(\epsilon_x + \epsilon_y \right)$$

$$\tau_{xy} = \frac{E}{2(1 + \nu)} \gamma_{xy}$$

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