

Plane Geometry

Circle

Circumference = $2\pi r$
Area = πr^2



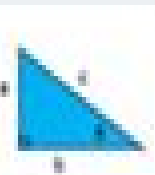
Parallelogram

Area = bh



Right Triangle


$a^2 + b^2 = c^2$
 $\sin \theta = \frac{a}{c}$
 $\cos \theta = \frac{b}{c}$
 $\tan \theta = \frac{a}{b}$



Solid Geometry

Cube

Volume = s^3
Surface Area = $6s^2$



Rectangular Prism

Volume = whl
Surface Area = $2(wh + wl + hl)$



Right Circular Cone

Volume = $\frac{\pi r^2 h}{3}$
Surface Area = $\pi r \sqrt{r^2 + h^2}$



Pyramid

Volume = $\frac{Ah}{3}$
A = area of base



Ellipse

Area = πab




Rectangle

Perimeter = $2a + 2b$
Area = ab



Triangle

Area = $\frac{1}{2}bh$
 $a^2 = b^2 + c^2 - 2bc \cos A$
 $b^2 = a^2 + c^2 - 2ac \cos B$
 $c^2 = a^2 + b^2 - 2ab \cos C$



Regular Polygon

Area = $\frac{ns^2}{4 \tan(\frac{\pi}{n})}$
n = number of sides




Trapezoid

Area = $\frac{1}{2}(a+b)h$




Sphere

Volume = $\frac{4}{3}\pi r^3$
Surface Area = $4\pi r^2$




Cylinder

Volume = $\pi r^2 h$
Surface Area = $2\pi r^2 + 2\pi rh$



Angular Prism

Volume = As
A = area of base



Constants

$g = 9.8 \text{ m/s}^2 = 32.17 \text{ ft/s}^2$
 $G = 6.67 \times 10^{-11} \text{ m}^3/\text{kg s}^2$
 $\pi = 3.14159$

Material Properties

Stress (axial)

$\sigma = \frac{F}{A}$

σ = stress
F = axial force
A = cross-sectional area

Strain (axial)

$\epsilon = \frac{\Delta L}{L_0}$

ϵ = strain
 L_0 = original length
 Δ = change in length

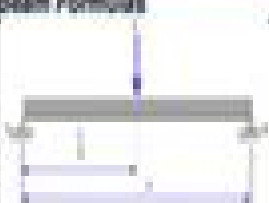
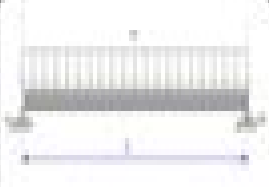


Modulus of Elasticity

$E = \frac{\sigma}{\epsilon}$

$E = \frac{dF/dL}{(L_1 - L_0)/L_0}$

E = modulus of elasticity
 σ = stress
 ϵ = strain
 A = cross-sectional area
 F = axial force
 Δ = deformation

Structural Analysis

| Beam Formula | Reaction | Moment | Deflection |
|---|---|--|---|
|  | $R_1 = R_2 = \frac{P}{2}$ | $M_{max} = \frac{Pa}{2}$ (at point of load) | $\Delta_{max} = \frac{Pa^2}{6EI}$ (at point of load) |
|  | $R_1 = R_2 = \frac{wL}{2}$ | $M_{max} = \frac{wL^2}{8}$ (at center) | $\Delta_{max} = \frac{5wL^4}{384EI}$ (at center) |
|  | $R_1 = R_2 = P$ | $M_{max} = P_1 a$ (between loads) | $\Delta_{max} = \frac{P_1 a^2}{24EI} (3L - 4a^2)$ (at center) |
|  | $R_1 = \frac{Pb}{L}$ and $R_2 = \frac{Pa}{L}$ | $M_{max} = \frac{Pab}{L}$ (at Point of Load) | $\Delta_{max} = \frac{Pab(L^2 - a^2 - b^2)}{24EI}$ (at $x = \sqrt{\frac{a^2 b^2}{L}}$ when $a > b$) |

Deformation: Axial

$\Delta = \frac{FL_0}{AE}$

Δ = deformation
 F = axial force
 L_0 = original length
 A = cross-sectional area
 E = modulus of elasticity

Truss Analysis

$2J = M + R$

J = number of joints
 M = number of members
 R = number of reaction forces

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